

Rayleigh Damping

The $n \times n$ symmetric damping matrix $[C]$ is formulated as a linear combination of the mass $[M]$ and stiffness $[K]$ matrices:

$$[C] = \alpha[M] + \beta[K] \quad (\text{Equation 1})$$

- **Alpha Coefficient:** Sets the mass-proportional coefficient α .
- **Beta Coefficient:** Sets the stiffness-proportional coefficient β .

The type of damping described by (Equation 1) is known as Rayleigh or proportional damping.

This form of $[C]$ is orthogonal with respect to the system eigenvectors.

By applying the modal coordinate transformation, the modal damping matrix $[c]$ becomes diagonal:

$$[\Phi]^T[C][\Phi] = [c] = \alpha[1] + \beta[\omega^2] \quad (\text{Equation 2})$$

You can define Rayleigh damping for linear and nonlinear dynamic studies.

Relation of Rayleigh Coefficients and Modal Damping Ratio

The modal damping matrix $[c]$ is given by:

$$[c] = 2[\zeta \omega] \quad (\text{Equation 3})$$

The coefficient of viscous damping c_i for the i th mode is calculated by:

$$c_i = 2 \zeta_i \omega_i = \alpha + \beta \omega_i^2 \quad (\text{Equation 4}),$$

and the viscous damping ratio ζ_i is expressed as

$$\zeta_i = \alpha / (2\omega_i) + \beta \omega_i / 2 \quad (\text{Equation 5})$$

If the damping ratios for the i th and j th modes are ζ_i and ζ_j , then the Rayleigh coefficients α and β are calculated from the solution of the two algebraic equations:

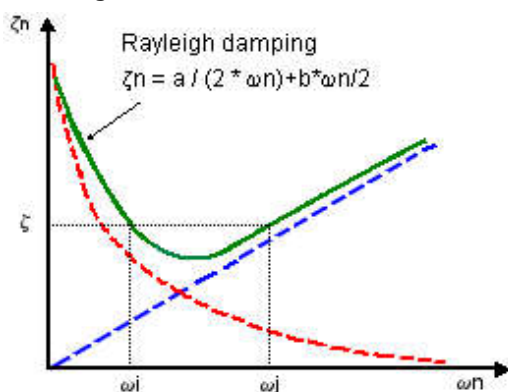
$$\frac{1}{2} \begin{bmatrix} 1 / \omega_i & \omega_i \\ 1 / \omega_j & \omega_j \end{bmatrix} \begin{Bmatrix} \alpha \\ \beta \end{Bmatrix} = \begin{Bmatrix} \zeta_i \\ \zeta_j \end{Bmatrix} \quad (\text{Equation 6})$$

If both modes have the same damping ratio ($\zeta_i = \zeta_j = \zeta$), then the values of α and β are given by:

$$\alpha = \zeta \frac{2\omega_i\omega_j}{\omega_i + \omega_j} \quad (\text{Equation 7})$$

$$\beta = \zeta \frac{2}{\omega_i + \omega_j} \quad (\text{Equation 8})$$

The viscous damping ratio ζ for any other mode varies with frequency as shown in the figure:



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